Statics and Strength of Materials Formula Sheet

(12/12/94, revised 5/10/01, 12/14/02 — A. Ruina)

Not given here are the conditions under which the formulae are accurate or useful.

Basic Statics

Free Body Diagram

A **FBD** is a picture of any system for which you would like to apply mechanics equations and of all the external forces and torques which act on the system.

Action & Reaction

If $\mathcal{A}$ feels force $\vec{\mathbf{F}}$ and couple $\vec{\mathbf{M}}$ from $\mathcal{B}$.
then $\mathcal{B}$ feels force $-\vec{\mathbf{F}}$ and couple $-\vec{\mathbf{M}}$ from $\mathcal{A}$.
(With $\vec{\mathbf{F}}$ and $-\vec{\mathbf{F}}$ acting on the same line of action.)

Force and Moment Balance

These equations apply to every system in equilibrium:

$$\overbrace{\sum_{\substack{\text{All external}\\ \text{forces}}} \vec{\mathbf{F}} = \vec{\mathbf{0}}}^{\textit{Force Balance}} \qquad \overbrace{\sum_{\substack{\text{All external}\\ \text{torques}}} \vec{\mathbf{M}}_{/C} = \vec{\mathbf{0}}}^{\textit{Moment Balance about pt C}}$$

- The torque $\vec{\mathbf{M}}_{/C}$ of a force depends on the reference point C. But, for a body in equilibrium, and for any point C, the sum of all the torques relative to point C must add to zero).
- Dotting the force balance equation with a unit vector gives a scalar equation,
  e.g. $\left\{\sum \vec{\mathbf{F}}\right\} \cdot \hat{\imath} = 0 \quad \Rightarrow \quad \sum F_x = 0.$
- Dotting the moment balance equation with a unit vector gives a scalar equation, e.g.,
  $$\left\{\sum \vec{\mathbf{M}}_{/C}\right\} \cdot \hat{\lambda} = 0$$
  $\Rightarrow$ net moment about axis in direction $\hat{\lambda}$ through $C = 0$.

Some Statics Facts and Definitions

- The moment of a force is unchanged if the force is slid along its line of action.
- For many purposes the words 'moment', 'torque', and 'couple' have the same meaning.
- **Two-force body.** If a body in equilibrium has only two forces acting on it then the two forces must be equal and opposite and have a common line of action.
- **Three-force body.** If a body in equilibrium has only three forces acting on it then the three forces must be coplanar and have lines of action that intersect at one point.
- **truss:** A collection of weightless two-force bodies connected with hinges (2D) or ball and socket joints (3D).
- **Method of joints.** Draw free body diagrams of each of the joints in a truss.
- **Method of sections.** Draw free body diagrams of various regions of a truss. Try to make the FBD cuts for the sections go through only three bars with unknown forces (2D).
- **Caution:** Machine and frame components are often **not** two-force bodies.
- **Hydrostatics:** $p = \rho g h, \quad F = \int p\, dA$

Miscellaneous

- **Power in a shaft:** $P = T\omega$.
- **Saint Venant's Principle:** Far from the region of loading, the stresses in a structure would only change slightly if a load system were replaced with any other load system having the same net force and moment.

Cross Section Geometry

	Definition	Composite	annulus (circle: $c_1 = 0$)	thin-wall annulus (approx)	rectangle
A	$\int dA$	$\sum A_i$	$\pi(c_2^2 - c_1^2)$	$2\pi ct$	bh
J I_p	$\int \rho^2\, dA$		$\frac{\pi}{2}(c_2^4 - c_1^4)$	$2\pi c^3 t$	
I	$\int y^2\, dA$	$\sum (I_i + d_i^2 A_i)$	$\frac{\pi}{4}(c_2^4 - c_1^4)$	$\pi c^3 t$	$bh^3/12$
$\bar{y}$	$\frac{\int y\, dA}{\int dA}$	$\frac{\sum y_i A_i}{\sum A_i}$	center	center	center

Stress, strain, and Hooke's Law

	Stress	**Strain**	**Hooke's Law**
Normal:	$\sigma = P_\perp / A$	$\epsilon = \delta / L_0 = \frac{L - L_0}{L_0}$	$\sigma = E\epsilon$ $[\,\epsilon = \sigma/E + \alpha \Delta T\,]$ $\epsilon_{tran} = -\nu \epsilon_{long}$
Shear:	$\tau = P_\parallel / A$	$\gamma =$ change of formerly right angle	$\tau = G\gamma$ $2G = \frac{E}{1+\nu}$

Stress and deformation of some things

	Equilibrium	Geometry	Results
Tension	$P = \sigma A$	$\epsilon = \delta / L$	$\delta = \frac{PL}{AE}$ $\delta = \frac{PL}{AE} + \alpha L \Delta T$
Torsion	$T = \int \rho \tau\, dA$	$\gamma = \rho \phi / L$	$\phi = \frac{TL}{JG}$ $\tau = \frac{T\rho}{J}$
Bending in Beams	$M = -\int y \sigma\, dA$ $\frac{dM}{dx} = V$ $\frac{dV}{dx} = -q$	$\epsilon = -y/\rho = -y\kappa$ $v'' = \frac{d^2}{dx^2} v = \frac{1}{\rho} = \kappa$	$v'' = \frac{M}{EI}$ $\sigma = \frac{-My}{I}$