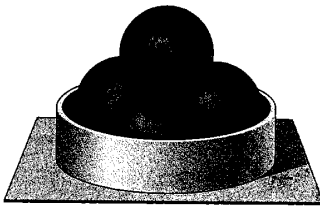


TAM 202 Spring03 HW 9 Soln provided by Peeyush and Tian
3/108, 6/131, 5/189, 6/59 (due 03/25/03)

3/108 Three identical steel balls, each of mass m , are placed in the cylindrical ring which rests on a horizontal surface and whose height is slightly greater than the radius of the balls. The diameter of the ring is such that the balls are virtually touching one another. A fourth identical ball is then placed on top of the three balls. Determine the force P exerted by the ring on each of the three lower balls.

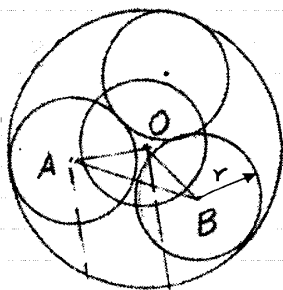


Problem 3/108

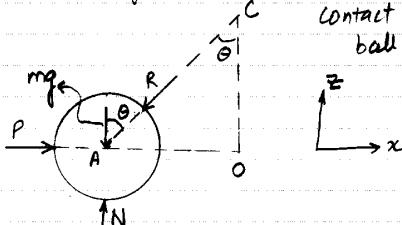
Solution:

Top view →

Angles OAB and ABD are 30° .



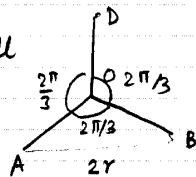
FBD for the ball A (typical). Note, assume no contact between ball A, B, D



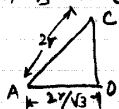
Geometry

C is the centre of the upper ball
Length of $AB = \bar{AB} = 2r = \bar{AC}$

From ΔADB , $\bar{AD} = \frac{r}{\cos 30^\circ} = \frac{2r}{\sqrt{3}}$.

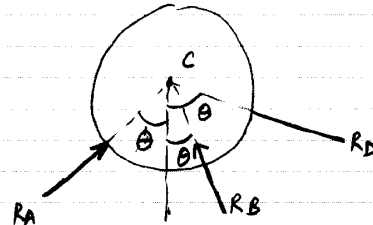


From ΔAOC , $\bar{OC} = \sqrt{AC^2 - OA^2} = r\sqrt{2^2 - (2/\sqrt{3})^2}$



or $\bar{OC} = 2r\sqrt{\frac{2}{3}}$ (continued)

FBD for the upper (top) ball



Equilibrium implies, $\sum F_z = 0$.

so $\sum F_z = 3R \cos \theta - mg = 0$

or $3R \left(\frac{\bar{OC}}{\bar{AC}} \right) - mg = 0$ [$\cos \theta = \frac{\bar{OC}}{\bar{AC}}$ from ΔOAC]

or $R = \frac{mg \cdot 2r}{2r\sqrt{\frac{2}{3}}} \cdot \frac{1}{3} = \frac{mg}{\sqrt{6}}$

Equilibrium for ball (see FBD), ball A

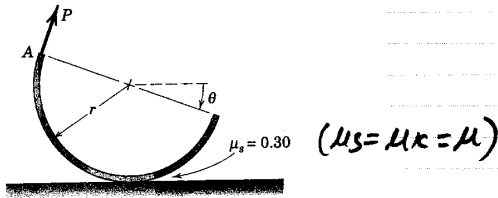
$\sum F_x = 0 \Rightarrow P - R \sin \theta = 0$.

or $P = R \sin \theta = \frac{mg}{\sqrt{6}} \frac{2r/\sqrt{3}}{2r} = \frac{mg}{3\sqrt{2}}$

so $P = \frac{mg}{3\sqrt{2}}$ lb.

*6/131 The semicylindrical shell of mass m and radius r is rolled through an angle θ by the force P which remains tangent to its periphery at A as shown. If P is slowly increased, plot the tilt angle θ as a function of P up to the point of slipping. Determine the tilt angle θ_{max} and the corresponding value P_{max} for which slipping occurs. The coefficient of static friction is 0.30.

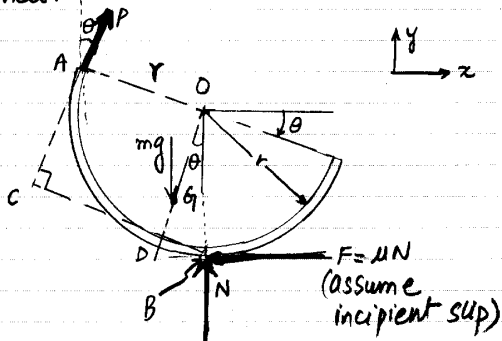
Ans. $\theta_{max} = 59.9^\circ$
 $P_{max} = 0.295mg$



Problem 6/131

Solution:

FBD for the shell:



$\bar{OG} = d = \frac{2r}{\pi}$; (See Table D/14, Pg 285 Meriam & Kraige)
 $\mu = 0.3$

Equilibrium:

$$\sum F_x = 0 \Rightarrow P \sin \theta - F = 0 \quad (1)$$

$$\sum F_y = 0 \Rightarrow N - mg + P \cos \theta = 0 \quad (2)$$

$$\sum M_B = 0 \Rightarrow mg d \sin \theta - P(\bar{BC}) \quad (3)$$

$$\bar{BC} = \bar{BD} + \bar{DC} = r + r \sin \theta.$$

At the start of slipping $F = \mu N$. (4)

(continued)

Relation between P & θ can be derived from (3)

$$P = \frac{2 \sin \theta * (mg)}{\pi(1 + \sin \theta)} \quad (5)$$

Substitute $F = \mu N$ in (1), we get

$$P \sin \theta = \mu N \Rightarrow N = \frac{P \sin \theta}{\mu} \quad (6)$$

Substitute $N = \frac{P \sin \theta}{\mu}$ in (2) to find relation between

P and mg .

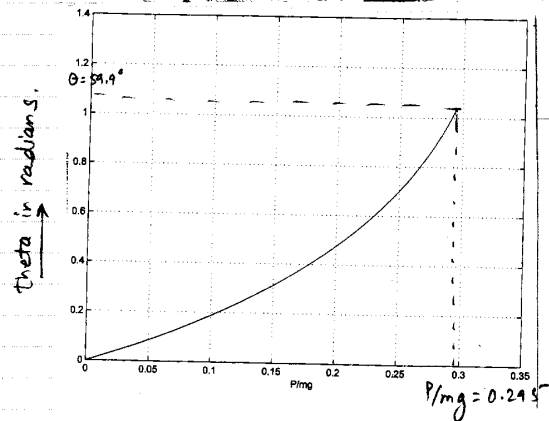
$$\frac{P \sin \theta}{\mu} - mg + P \cos \theta = 0 \Rightarrow P = \frac{\mu mg}{\mu \cos \theta + \sin \theta} \quad (7)$$

Comparing (5) & (7)

$$\frac{\mu}{\mu \cos \theta + \sin \theta} = \frac{2 \sin \theta}{\pi(1 + \sin \theta)} \quad (8)$$

solve (8) numerically to obtain $\theta_{max} = 59.9^\circ$

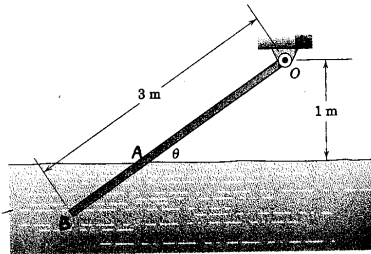
& corresponding $P_{max} = 0.295 mg$ using (5).



Matlab Program.

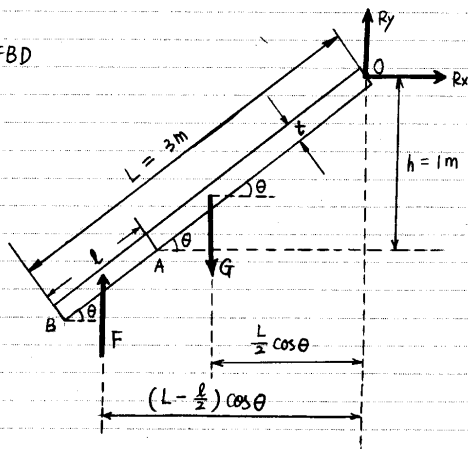
```
theta=0:0.1:(59.9*pi/180);
P=2.*sin(theta)./(pi.*(1+sin(theta)));
plot(P,theta)
xlabel('P/mg')
ylabel('theta(in radians)')
grid on
```

5/189 The 3-m plank shown in section has a density of 800 kg/m³ and is hinged about a horizontal axis through its upper edge O. Calculate the angle θ assumed by the plank with the horizontal for the level of fresh water shown.
 Ans. $\theta = 48.2^\circ$



Problem 5/189

① FBD



Assume:

width of the plank (into the paper) = b

thickness of the plank = t

then

$V_p \equiv$ Volume of the plank = Lbt

$V_s \equiv$ Volume of the part of the plank submerged into the water

= lbt

$\Rightarrow G = \rho_p g V_p = \rho_p g Lbt$

where ρ_p is the density of the plank.

(Continued)

Buoyancy $F = \rho_w g V_s = \rho_w g lbt$

where ρ_w is the density of the water

② Equilibrium eqn.

$$\sum M_{10} = 0 = G \left(\frac{1}{2} \cos \theta\right) + F \left(L - \frac{l}{2}\right) \cos \theta$$

$$= \rho_p g Lbt \left(\frac{1}{2} \cos \theta\right) - \rho_w g lbt \left(L - \frac{l}{2}\right) \cos \theta$$

$$\Rightarrow \rho_p L^2 - \rho_w l(2L - l) = 0$$

$$\Rightarrow l^2 - 2Ll + \frac{\rho_p}{\rho_w} L^2 = 0$$

substitute numbers in:

$$l^2 - 2(3m)l + \frac{800 \text{ kg/m}^3}{1000 \text{ kg/m}^3} (3m)^2 = 0$$

$$\Rightarrow l = \frac{2(3m) \pm \sqrt{(6m)^2 - 4(4/5)(3m)^2}}{2}$$

$$= 1.66 \text{ m} \quad \text{or} \quad 4.34 \text{ m (rejected)}$$

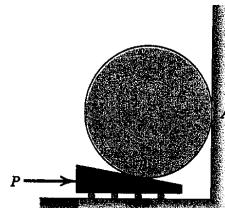
$$\Rightarrow l = L - \frac{h}{\sin \theta} = 1.66 \text{ m}$$

$$\Rightarrow 3m - \frac{1m}{\sin \theta} = 1.66 \text{ m}$$

$$\Rightarrow \theta = 48.2^\circ$$

6/59 Calculate the horizontal force P on the light 10° wedge necessary to initiate movement of the 40-kg cylinder. The coefficient of static friction for both pairs of contacting surfaces is 0.25. Also determine the friction force F_B at point B. (Caution: Check carefully your assumption of where slipping occurs.)

Ans. $P = 98.6 \text{ N}$, $F_B = 24.6 \text{ N}$

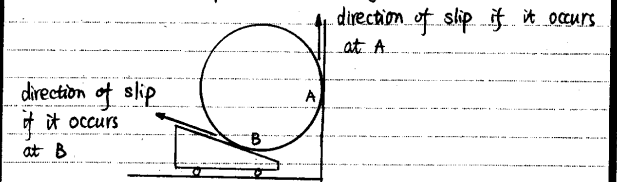


Problem 6/59

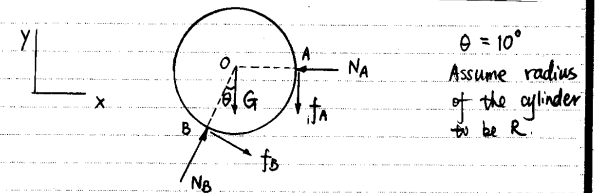
(Continued)

① FBD of the cylinder.

First, we should determine the directions of friction forces at A & B. Since we are pushing the wedge at A OR slip outward at B (we don't know whether it will slip at A or B yet), i.e.



Since friction force is in the opposite direction to the slip, we get the FBD of the cylinder.



$\theta = 10^\circ$
Assume radius of the cylinder to be R.

② Equilibrium Eqns:

$$\sum M_{10} = 0 = f_B R - f_A R \Rightarrow f_A = f_B \quad (1)$$

$$\sum F_x = 0 = N_B \sin \theta + f_B \cos \theta - N_A \quad (2)$$

$$\sum F_y = 0 = N_B \cos \theta - f_B \sin \theta - f_A - G \quad (3)$$

Now we have 3 eqns but 4 unknowns

(N_A , N_B , f_A , f_B), so we need one more eqn.

This is the equation relating friction force and normal force when the cylinder starts to slip, i.e. this eqn is either $f_A = \mu N_A$ or $f_B = \mu N_B$, depending on at which point (A or B) the cylinder slips. Since we don't know at this time, let's try.
(Continued)

If it slips at B

$$f_B = \mu N_B \quad \text{sub into (2)} \Rightarrow N_B \sin \theta + \mu N_B \cos \theta - N_A = 0$$

$$\Rightarrow N_A = N_B (\sin \theta + \mu \cos \theta) = N_B (\sin 10^\circ + 0.25 \cos 10^\circ) = 0.42 N_B$$

$$\text{From (1)} \Rightarrow f_A = f_B = \mu N_B$$

$$\Rightarrow \frac{f_A}{N_A} = \frac{\mu N_B}{0.42 N_B} = \frac{\mu}{0.42} > \mu$$

\Rightarrow When B slips, the friction at A already exceeds $\mu N_A \Rightarrow$ Bad assumption!

So the slip should first occur at point A. Let's check:

If it slips at A

$$f_A = \mu N_A \quad \text{from (1)} \Rightarrow f_B = \mu N_A$$

$$\text{sub into (2)} \Rightarrow N_B \sin \theta + \mu N_A \cos \theta - N_A = 0$$

$$\Rightarrow N_B = \frac{N_A (1 - \mu \cos \theta)}{\sin \theta} = \frac{N_A (1 - 0.25 \cos 10^\circ)}{\sin 10^\circ} = 4.34 N_A$$

$$\Rightarrow \frac{f_B}{N_B} = \frac{\mu N_A}{4.34 N_A} = \frac{\mu}{4.34} < \mu$$

\Rightarrow when A slips, B doesn't slip

\Rightarrow No paradox!

So the slip first occurs at A.

$$\text{Another equation: } f_A = \mu N_A \quad (4)$$

Solve (1) ~ (4):

$$\text{Sub (4) into (1)} \Rightarrow f_B = \mu N_A \quad (5)$$

$$\text{sub (5) into (2)} \Rightarrow N_B = \frac{N_A (1 - \mu \cos \theta)}{\sin \theta} \quad (6)$$

sub (5), (6) into (3)

$$\Rightarrow \cos \theta \frac{N_A (1 - \mu \cos \theta)}{\sin \theta} - \mu N_A \sin \theta - \mu N_A - G = 0$$

$$\Rightarrow N_A = \frac{G \sin \theta}{\cos \theta - \mu (1 + \sin \theta)}$$

$$= \frac{(40 \text{ kg})(9.81 \text{ N/kg}) \sin 10^\circ}{\cos 10^\circ - 0.25(1 + \sin 10^\circ)} \Rightarrow \boxed{N_A = 98.6 \text{ N}}$$

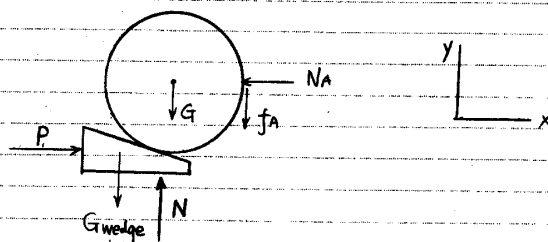
(Continued)

$$\text{Sub } N_A = 98.6 \text{ N into (5)}$$

$$\Rightarrow \boxed{f_B = 0.25(98.6 \text{ N}) = 24.6 \text{ N}}$$

③ P

FBD of the "cylinder + wedge" system



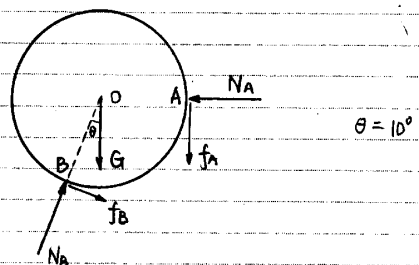
$$\Sigma F_x = 0 = P - N_A$$

$$\Rightarrow \boxed{P = N_A = 98.6 \text{ N}}$$

Comments

One can also use a graphic method to determine whether the cylinder will slip at A or B.

First, the FBD of the cylinder is

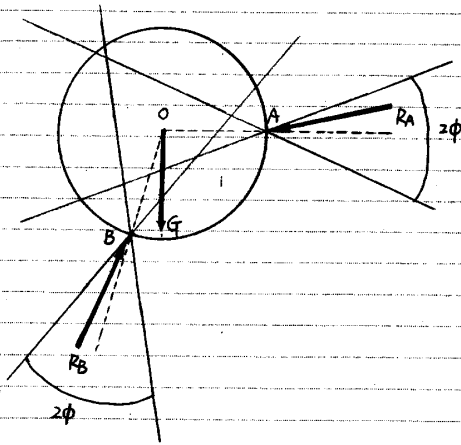


Second, for the equilibrium of the cylinder, the resultant of f_A and N_A (denoted as R_A) can only lie in a wedge at A with angle 2ϕ , where

$$\phi = \tan^{-1} \mu = \tan^{-1}(0.25) \approx 14^\circ$$

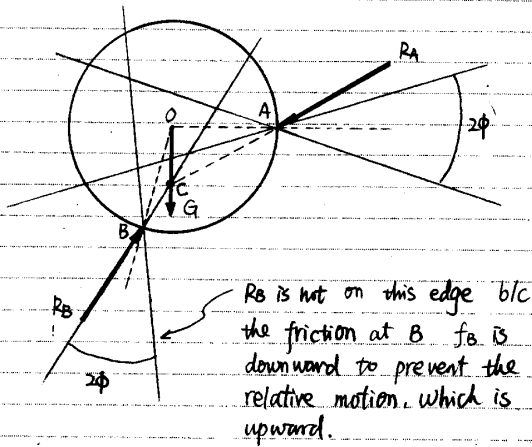
For the same reason, the resultant of f_B and N_B (denoted as R_B) can also only lie in a wedge at B with same angle 2ϕ (See figure below)

(Continued)



Third, the cylinder is a 3-force member (G , R_A and R_B), so these three forces have to pass through a same point. Now let's see whether A or B will slip first.

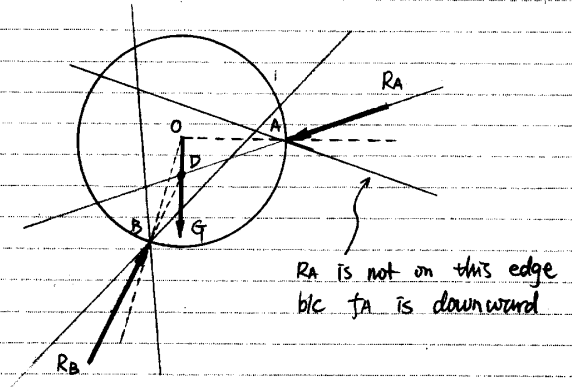
If B slips first, then R_B is on the edge of the wedge, i.e.



R_B is not on this edge b/c the friction at B is downward to prevent the relative motion, which is upward.

So R_B and G intersect at point C. For equilibrium, R_A must pass through C too, as shown. But then R_A doesn't lie in the wedge any more, which is a paradox. (Continued)

If A slips first, then R_A is on the edge of the wedge, i.e.



R_A is not on this edge b/c f_A is downward.

R_A and G intersect at D, so R_B must pass through D to maintain equilibrium, as shown above. We can see that R_B is still in the wedge, so there is no paradox, and we get the conclusion that A must slip first.