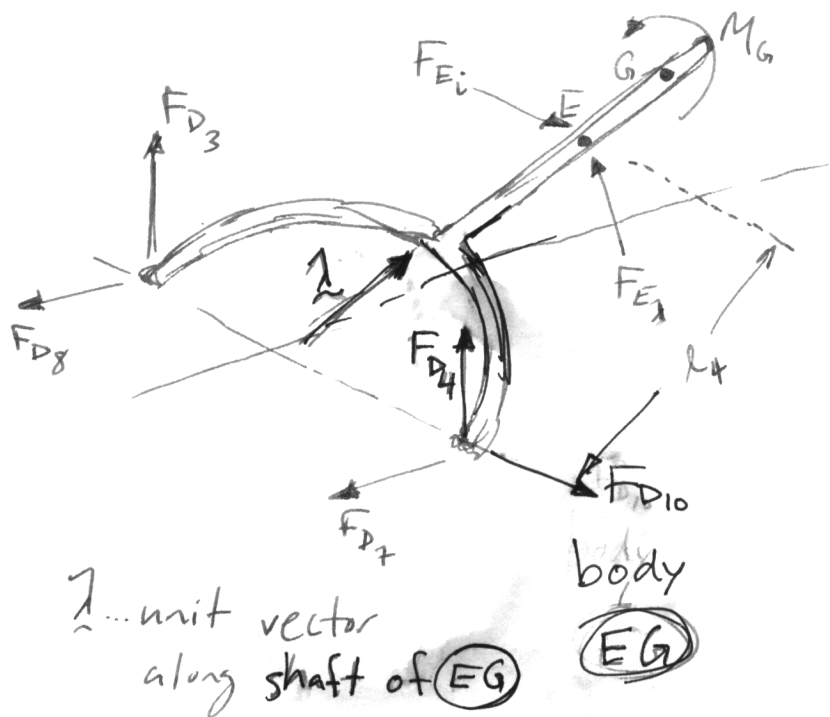
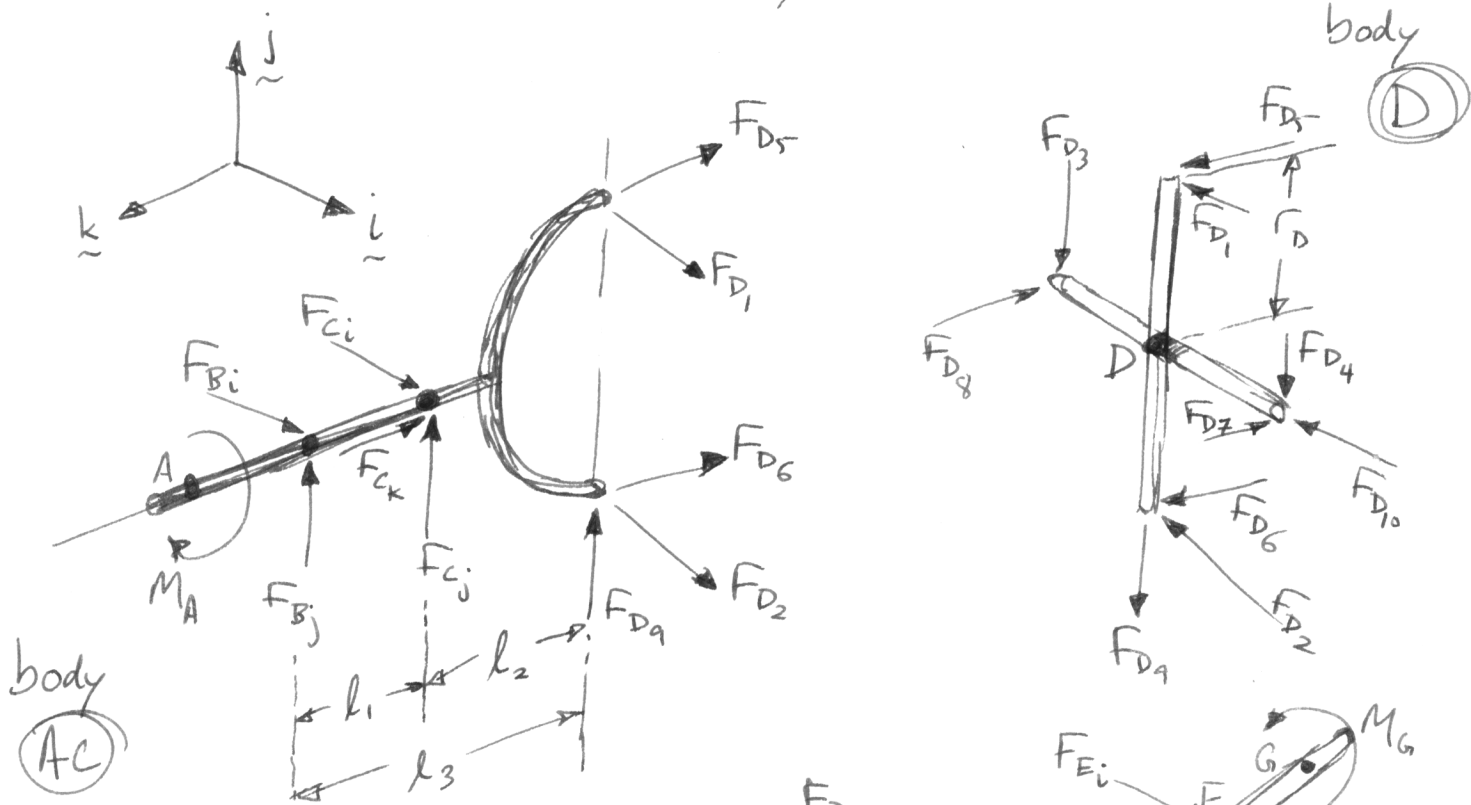


Problem #1, parts (a) and (b)

MAE 325 Fall 1999
 HW3 Solution
 Due wed 9/15/99

1/10

Strategy: FBD, derive all equations, reduce equations by choosing particular components with order of reduction chosen wisely



$$M_A = 30 \text{ Nm}$$

$$l_1 = 0.1 \text{ m}$$

$$l_2 = 0.125 \text{ m}$$

$$l_3 = l_1 + l_2 = 0.225 \text{ m}$$

$$l_4 = 0.15 \text{ m}$$

$$\lambda = \sin \delta \hat{j} + \cos \delta \hat{k}$$

$$\delta = 25^\circ$$

λ ... unit vector along shaft of (EG)

Now have FBD of all components and 18 unknowns, 2/10

need 18 equations ... three from each of angular and linear momentum balance on the three components.

Moment equations come from vector cross products, with only the result given below.

in (A)

$$\begin{aligned}\sum \underline{M}_{/B} = \underline{0} = & \underline{i} (l_1 F_{Cj} + l_3 F_{D_9} + r_D F_{D_6} - r_D F_{D_5}) \\ & + \underline{j} (-l_1 F_{Ci} - l_3 F_{D_1} - l_3 F_{D_2}) \\ & + \underline{k} (-M_A - r_D F_{D_1} + r_D F_{D_2})\end{aligned}$$

$$\begin{aligned}\sum \underline{F} = \underline{0} = & \underline{i} (F_{B_i} + F_{C_i} + F_{D_1} + F_{D_2}) + \underline{j} (F_{B_j} + F_{C_j} + F_{D_9}) \\ & + \underline{k} (-F_{C_k} - F_{D_5} - F_{D_6})\end{aligned}$$

in (D)

$$\begin{aligned}\sum \underline{M}_{/D} = \underline{0} = & r_D \underline{k} (F_{D_1} - F_{D_2} + F_{D_3} - F_{D_4}) + r_D \underline{i} (F_{D_5} - F_{D_6}) \\ & + r_D \underline{j} (F_{D_7} - F_{D_8})\end{aligned}$$

$$\begin{aligned}\sum \underline{F} = \underline{0} = & \underline{i} (-F_{D_1} - F_{D_2} - F_{D_{10}}) + \underline{j} (-F_{D_3} - F_{D_4} - F_{D_9}) \\ & + \underline{k} (F_{D_5} + F_{D_6} - F_{D_7} - F_{D_8})\end{aligned}$$

(EG)

$$\begin{aligned}\sum \underline{M}_{/E} = \underline{0} = & \underline{i} [-l_4 \cos \gamma (F_{D_3} + F_{D_4}) - l_4 \sin \gamma (F_{D_7} + F_{D_8})] \\ & + \underline{j} [-M_G \sin \gamma + l_4 \cos \gamma F_{D_{10}} + r_D (F_{D_8} - F_{D_7})] \\ & + \underline{k} [M_G \cos \gamma + r_D (F_{D_4} - F_{D_3}) + l_4 \sin \gamma F_{D_{10}}]\end{aligned}$$

$$\underline{\Sigma F} = \underline{0} = \underline{i}(F_{D10} + F_{Ei}) + \underline{j}(F_{D3} + F_{D4} + F_{E\lambda} \cos \gamma) + \underline{k}(F_{D7} + F_{D8} + F_{E\lambda} \sin \gamma) \quad \boxed{3/10}$$

Choice of which equations to consider first can make it not too difficult to do by hand:

$$\textcircled{AC} \quad \underline{\Sigma M}_{\sim/B} \cdot \underline{k} = 0 = -M_A + r_D (F_{D2} - F_{D1}) \quad \therefore M_A = r_D (F_{D2} - F_{D1}) \quad \textcircled{1}$$

$$\textcircled{EG} \quad \underline{\Sigma M}_{\sim/E} \cdot \underline{i} = 0 = -M_G + r_D (F_{D3} - F_{D4}) \cos \gamma + r_D (F_{D8} - F_{D7}) \sin \gamma \quad \textcircled{2}$$

$$\textcircled{D} \quad \underline{\Sigma M}_{\sim/D} \cdot \underline{j} = 0 = r_D (F_{D7} - F_{D8}) \quad \therefore F_{D7} = F_{D8} \quad \textcircled{3}$$

$$\textcircled{D} \quad \underline{\Sigma M}_{\sim/D} \cdot \underline{k} = 0 = r_D (F_{D1} - F_{D2} + F_{D3} - F_{D4})$$

$$\therefore F_{D1} - F_{D2} = F_{D3} - F_{D4} \quad \textcircled{4}$$

Substitute $\textcircled{3}$ & $\textcircled{4}$ into $\textcircled{2}$ to give

$$M_G = r_D \cos \gamma (F_{D2} - F_{D1}) \quad \text{and substitute } \textcircled{1} \text{ to get}$$

$$M_G = M_A \cos \gamma \quad \text{or} \quad \boxed{M_G = 27.19 \text{ Nm}}$$

Note: this result could have been gotten without writing the full equations on all components, but we'll be putting them to use momentarily for part \textcircled{b} so it was not wasted effort.

For part (b), use the remaining equations to determine the unknowns: 4/10

$$\textcircled{EG} \sum \underline{M}_{/E} \cdot \underline{j} = 0 = -M_G \sin \gamma + l_4 \cos \gamma F_{D_{10}} + r_D (F_{D_8} - F_{D_7}) \quad \rightarrow 0 \text{ by } \textcircled{3}$$

$$\therefore F_{D_{10}} = \frac{M_G \sin \gamma}{l_4 \cos \gamma} \approx 84.52 \text{ N}$$

$$\textcircled{EG} \sum \underline{F} \cdot \underline{i} = 0 = F_{D_{10}} + F_{Ei} \quad \therefore \boxed{F_{Ei} = -84.52 \text{ N}}$$

$$\textcircled{EG} \sum \underline{M}_{/E} \cdot \underline{i} = 0 = l_4 \cos \gamma (F_{D_4} + F_{D_3}) - l_4 \sin \gamma (F_{D_7} + F_{D_8})$$

$$\textcircled{EG} \sum \underline{F} \cdot \underline{j} = 0 = F_{D_4} + F_{D_3} + F_{Ej} \cos \gamma \quad \left. \begin{array}{l} \\ \\ \end{array} \right\} F_{Ej} = -\frac{1}{\cos \gamma} (F_{D_4} + F_{D_3})$$

$$\textcircled{EG} \sum \underline{F} \cdot \underline{k} = 0 = F_{D_7} + F_{D_8} + F_{Ej} \sin \gamma \quad = -\frac{1}{\sin \gamma} (F_{D_7} + F_{D_8})$$

these may be written in matrix form

$$\begin{bmatrix} l_4 \cos \gamma & -l_4 \sin \gamma \\ -\frac{1}{\cos \gamma} & +\frac{1}{\sin \gamma} \end{bmatrix} \begin{bmatrix} F_{D_4} + F_{D_3} \\ F_{D_7} + F_{D_8} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

and because the matrix is non-singular, the solution must be

$$F_{D_4} + F_{D_3} = 0 \quad \textcircled{6} \quad \text{and} \quad F_{D_7} + F_{D_8} = 0$$

$$\text{thus} \quad \boxed{F_{Ej} = 0}$$

$$\text{and from } \textcircled{3}, \quad F_{D_7} = F_{D_8} \quad \text{so} \quad F_{D_7} = F_{D_8} = 0 \quad \textcircled{7}$$

ⓓ $\sum \underline{F} \cdot \underline{j} = 0 = -F_{D9} - (F_{D4} + F_{D3})$ ^{obj 6} $\therefore F_{D9} = 0$ ⑧

Ⓐ $\sum M_{\underline{B}} \cdot \underline{i} = 0 = -l_1 F_{Ci} - l_3 (F_{D1} + F_{D2}) = 0$

ⓓ $\sum \underline{F} \cdot \underline{i} = 0 = -(F_{D1} + F_{D2}) - F_{D10} \therefore F_{D1} + F_{D2} = -F_{D10}$ ⑪ } $F_{Ci} = \frac{l_3}{l_1} F_{D10}$ ⑫

$\therefore F_{Ci} \cong 190.2 \text{ N}$

Ⓐ $\sum M_{\underline{B}} \cdot \underline{i} = 0 = l_1 F_{Cj} + l_3 F_{D9} + r_D (F_{D6} - F_{D5})$ ⑩

ⓓ $\sum M_{\underline{D}} \cdot \underline{i} = 0 = (F_{D5} - F_{D6}) r_D \therefore F_{D5} = F_{D6}$ ⑨

substitute ⑧ & ⑨ into ⑩ $\Rightarrow F_{Cj} = 0$ ⑬

ⓓ $\sum \underline{F} \cdot \underline{k} = 0 = F_{D5} + F_{D6} - (F_{D7} + F_{D8})$ ^{obj 7}

combine with ⑨ to get $F_{D5} = F_{D6} = 0$ ⑭

Ⓐ $\sum \underline{F} \cdot \underline{k} = 0 = -F_{Ck} - F_{D5} - F_{D6} \therefore F_{Ck} = 0$ ^{obj 14}

Ⓐ $\sum \underline{F} \cdot \underline{i} = 0 = F_{Bi} + F_{Ci} + F_{D1} + F_{D2}$

using ⑪ & ⑫, $F_{Bi} = F_{D10} (1 - \frac{l_3}{l_1}) \therefore F_{Bi} \cong -105.7 \text{ N}$

Ⓐ $\sum \underline{F} \cdot \underline{j} = 0 = F_{Bj} + F_{Cj} + F_{D9}$ ^{obj 13} ^{obj 8} $\therefore F_{Bj} = 0$

part (c)

6/10

$$P_{in} = P_{out}$$

$$M_A \omega_{AC} = M_G \omega_{EG}$$

$$\omega_{EG} = \frac{M_A \omega_{AC}}{M_G} = \boxed{11.03 \frac{\text{rev}}{\text{sec}}}$$

part (d)

$$\underline{\omega}_{EG} = \underline{\omega}_{AC} + \underline{\omega}_{D/AC} + \underline{\omega}_{EG/D}$$

$$\textcircled{1} \quad \omega_{EG} \underline{\lambda} = \omega_{AC} \underline{k} + \omega_{D/AC} \underline{j} + \omega_{EG/D} \underline{i}$$

$$\textcircled{1} \cdot \underline{k} \Rightarrow \omega_{EG} \cos \gamma = \omega_A$$

$$\therefore \omega_{EG} = \frac{\omega_{AC}}{\cos \gamma} = \boxed{11.03 \frac{\text{rev}}{\text{sec}}}$$

Note that this is exactly the same result as for part (c)

part (e)

combining results from parts (c) and (d)

$$M_G = M_A \cos \gamma \text{ in current configuration}$$

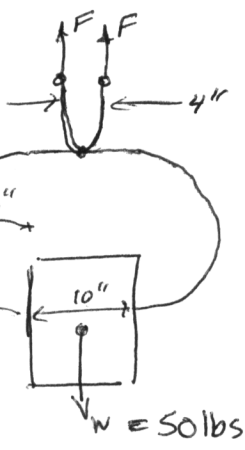
and for a rotation of 90°

$$\boxed{M_A = M_G \cos \gamma}$$

as the geometry of the problem as it relates to parts (c) and (d) is unchanged except for a reversal of the positions of input and output shafts

② p 3-17

FBD

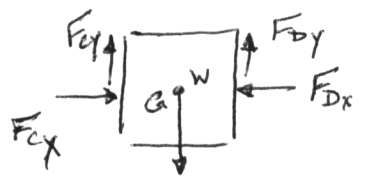


Find all Forces on the tongs.
Find the bending moment at A.

$$\sum F_y = 0 : 2F = W \quad (1)$$

$$F = \underline{\underline{25 \text{ lbs}}}$$

FBD



$$\sum F_y : F_{cy} + F_{dy} = W \quad (2)$$

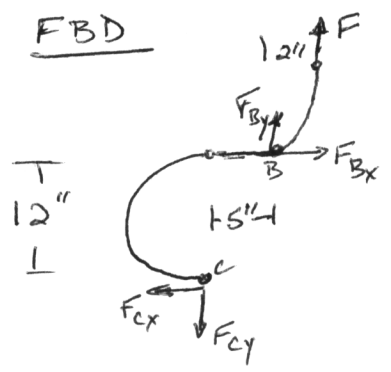
$$\sum M_G : F_{cy} = F_{dy} \quad (3)$$

Using (2), (3)

$$2F_{cy} = W$$

$$F_{cy} = \frac{1}{2}W = \underline{\underline{25 \text{ lbs}}}$$

FBD



$$\sum F_x : F_{Bx} - F_{cx} = 0$$

$$F_{Bx} = F_{cx} \quad (4)$$

$$\sum F_y : F_{By} - F_{cy} + F = 0 \quad (5)$$

$$F_{By} = F_{cy} - F$$

$$F_{By} = 25 - 25 = 0$$

$$\sum M_B : +12F_{cx} - 5F_{cy} - 2F = 0 \quad (6)$$

$$F_{cx} = \frac{1}{12} [5F_{cy} + 2F]$$

$$F_{cx} = \underline{\underline{14.17 \text{ lbs}}}$$

from (4)

$$F_{cx} = F_{Bx}$$

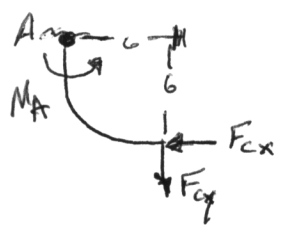
$$F_{Bx} = \underline{\underline{14.17 \text{ lbs}}}$$

FBD w/ cut

Summing moments

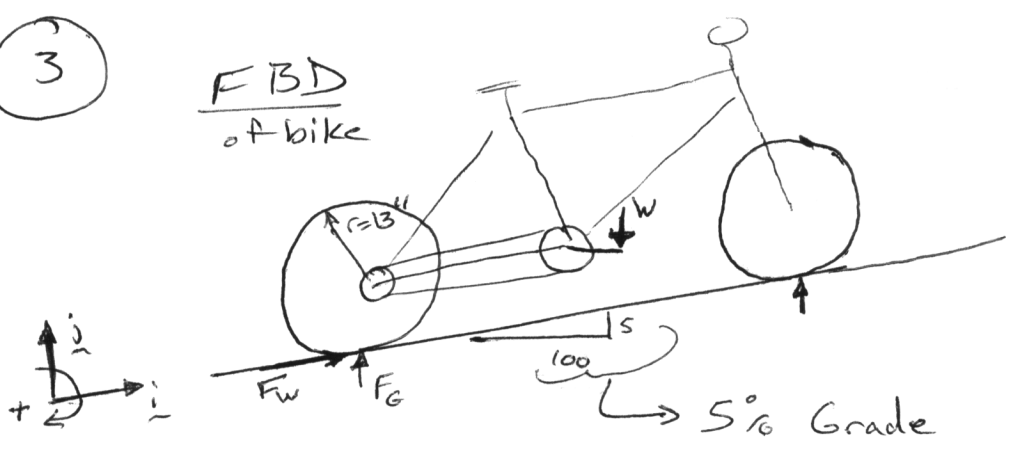
$$M_A = 6F_{cx} + 6F_{cy}$$

$$M_A = \underline{\underline{237.6 \text{ in lbs}}}$$



3

FBD of bike



Assume all of the riders weight is on the front pedal when it is horizontal and the bike + rider is in equilibrium going up a 5% grade. What is the ratio of teeth on the sprockets?



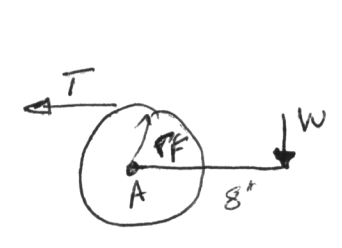
$$\tan \theta = \frac{5}{100}$$

$$\Rightarrow \theta = \tan^{-1} \frac{5}{100} = 2.86^\circ$$

$$\sum F_x: F_w - W \sin \theta = 0$$

$$F_w = W \sin \theta \quad (1)$$

FBD of Front Sprocket



$$\sum M_A: 8W = r_F T$$

$$T = \frac{8W}{r_F} \quad (2)$$

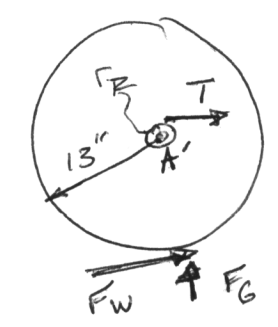
Subs. (2) into (3)

$$13 F_w = \frac{r_R}{r_F} 8W \quad (4)$$

Subs (1) into (4) and solve for tooth ratio

$$\frac{r_R}{r_F} = \frac{13 \sin \theta}{8 \sin \theta} = \underline{\underline{0.081}}$$

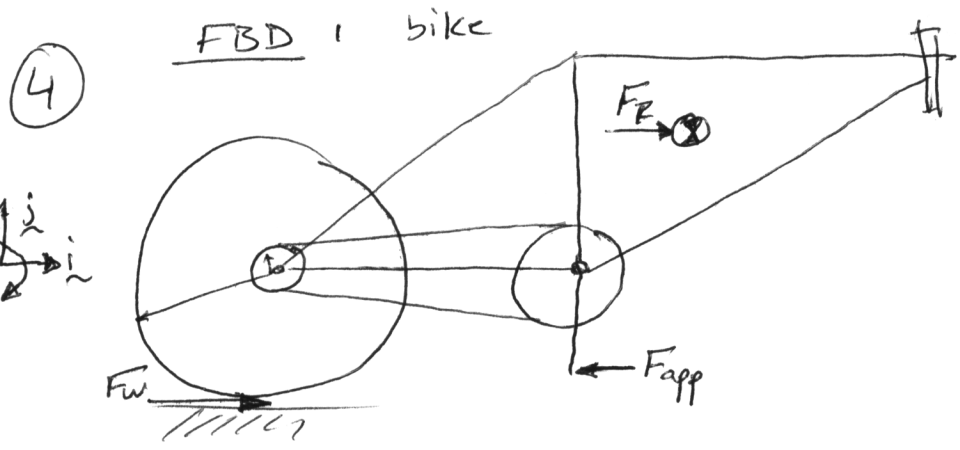
FBD of rear wheel/sprocket



$$\sum M_{A'}: 13 F_w = r_R T \quad (3)$$

$$\underline{\underline{\frac{r_F}{r_R} = 12.3}}}$$

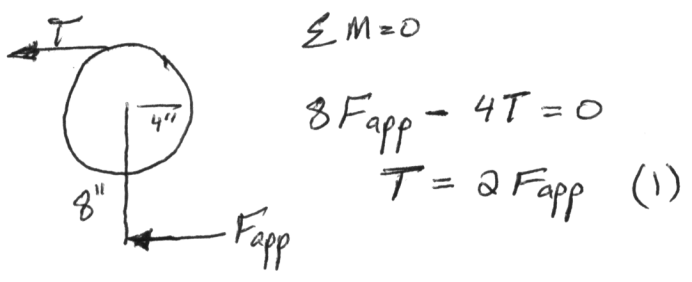
i.e. if the front sprocket has 100 teeth the rear sprocket has ~~8~~ teeth.



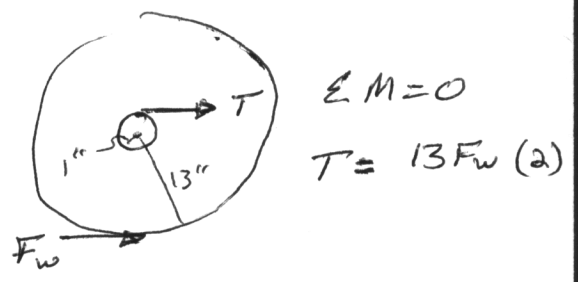
A person walks up to the bike and pushes on the bottom pedal w/ the crank vertical
 If the person pushes backwards which way does the bike go?

- wheel $r_w = 13''$
- rear sprocket $r_{rs} = 1''$
- front sprocket $r_{fs} = 4''$
- crank $r_c = 8''$

FBD 2 Front sprocket



FBD 3 Rear sprocket



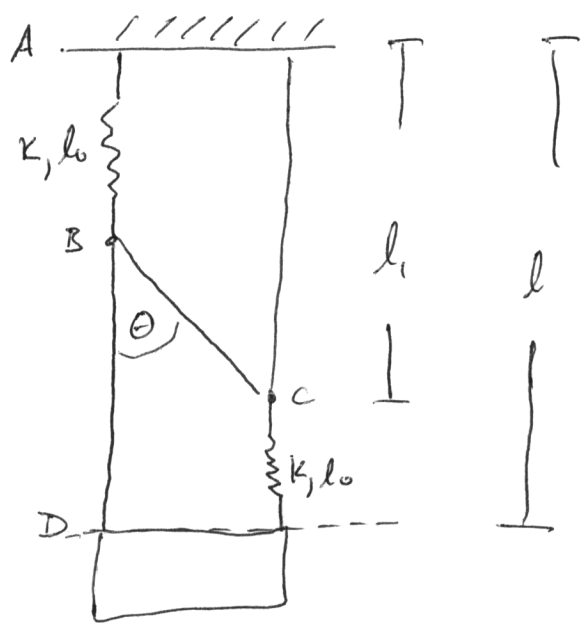
Now $\sum F_x = 0 \Rightarrow -F_{app} + F_w + F_R = 0 \quad (3)$

Using (1), (2) $\frac{2}{13} F_{app} = F_w \quad (4)$

Subs: (4) into (3) $-F_{app} + \frac{2}{13} F_{app} + F_R = 0$
 $F_R = \frac{11}{13} F_{app}$

This result means that a force of $\frac{11}{13} F_{app}$ (for this choice of dimensions) has to be applied to keep the bike in equilibrium without it, the bike would move backwards!

5



If all segments are taut, what happens when the string is cut?

For a spring:

$$F = kx$$

$$\text{so } l_0 = \frac{F}{k} \quad (1)$$

Look @ pt. B:



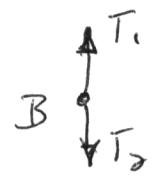
Assume $T > 0$ and $0 < \theta < 90$

$$\sum F_y: T_1 - T_2 - T \sin \theta = 0$$

$$T_1 = T_2 + T \sin \theta \quad (2)$$

some positive quantity

When the string is cut:



$$T_1 = T_2 \quad (3)$$

So with the string T_1 is greater than without it. This means that the springs would be stretched more w/ the ~~spring~~ crossmember BC than without (eq 1).

When the string is cut, the weight goes up (b) because the force on the springs is lower.